

Nomenclature of pulley with locking assembly			
Technical specification	Where		
Step 1: Application Torque calculation:	P _M = Motor Power , KW No = Motor speed , RPM i = Gear Box Ratio		
<i>Is calculated by resultant force formula and general torque formula. Ref, engineering hand book.</i>	T1 = Tight side Tension (N) T2 = Slack side Tension (N)		
Required Torque, $*Ma = \frac{k \times (T_1 - T_2) \times D_2}{2000} Nm$ $\frac{Or}{*MA} = \frac{P_M x \ 60 \ x \ 10^3}{2 \ x \ \pi \ x \frac{No}{i}}$	D2 = outer Diameter of Pulley (mm) k = start - up factor (on the basis of customer requirements) θ = Wrap angle in degree A _L = Lever arm length (mm) Ma = Required torque (Nm) Mba = Required honding Moment (Nm)		
* Required Torque, is considered whichever is maximum for further calculation.	Mtc = Composite torque (Nm)		
Step 2: Application Bending moment calculation :	Kb = 1.2 to 2 (Bending Moment factor, Depending upon your Application)		
By using force triangles, law of cosines and "Dubbel Handbook for Mechanical Engineer" bending moment formula required bending moment is calculated. Required bending moment, $Mba = (k \times AL) \times \frac{\sqrt{T_1^2 + T_2^2 - 2 \times T_1 \times T_2 \times \cos \theta}}{2000}$	 d = Shaft Diameter at Hub , mm d1 = Hub Outer diameter , mm L1 = Hub Length , mm T1 = Tight side Tension * start up Factor , N T2 = Slack side Tension * start up 		
2000	Factor , N		

Doc No · N-DD-LASCA	<u>Conveyor Application</u>		
Step 3:			
Combined torque calculation when there is			
simultaneous torque and bending load			
Composite Torque,			
$Mtc = \sqrt{M_a^2 + (kb \times M_b)}$	$(a)^2$		
For Drive Pulley,			
Catalogue Torque $Mt \ge I$	Mtc		
Catalogue <i>Mb≥Mba</i>			
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For Non-Drive Pulley,			
Catalogue <i>Mb≥Mba</i>			
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